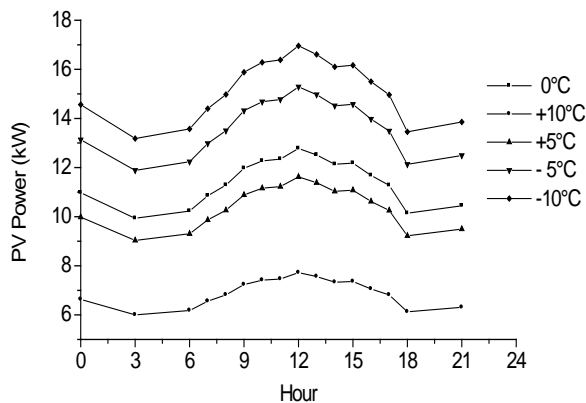


**Figure 12:** Hourly variation of PV power for the 1<sup>st</sup> days of the months from March to October 2012, for an evaporating temperature  $T_e=0^\circ\text{C}$

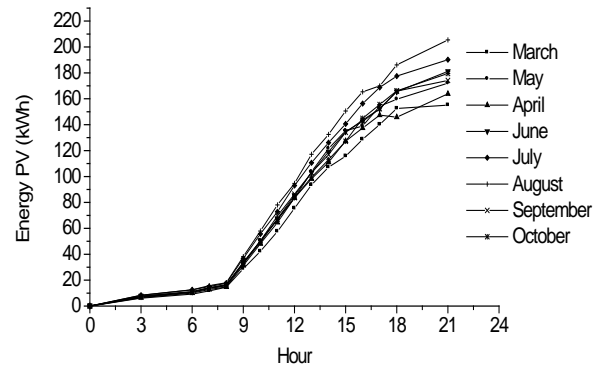
According to the results, it is higher in August with a value of 14.94 kWp and insolation of 4.86 Wh/m<sup>2</sup>/day. The minimum value is obtained in March at 3 am with 9.29 kWp and the maximum at 2pm (14.94 kWp) on August, while on April there is a minimum value of 9.93 kWp at 3 pm and a maximum of 12.77 kWp.

Figure 13 gives the hourly variation of the output of the PV modules for evaporating temperatures of -10 °C, -5 °C, 0 °C, 5 °C and 10 °C in April. It could be seen that it is low for an evaporating temperature of 10 °C and higher for an evaporating temperature of -10 °C. We could therefore conclude that when the evaporating temperature decreases as the power of PV modules increases.

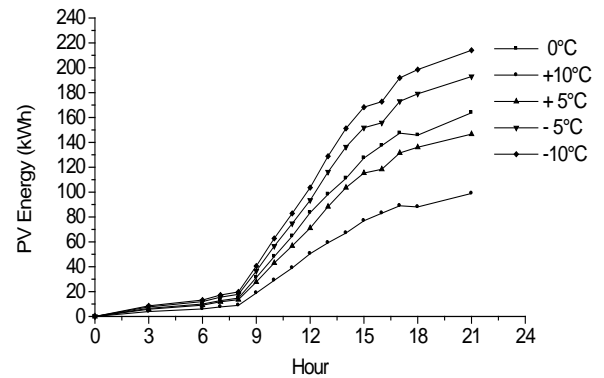


**Figure 13:** Hourly variation of PV power for evaporating temperatures of 0°C, +10°C, +5°C, -5°C and -10°C on 1<sup>st</sup> April 2012

After calculating the different powers and energies provided by the PV panels as shown on figures 14 and 15, we can make a comparison between them and the powers and energies absorbed by the compressor.

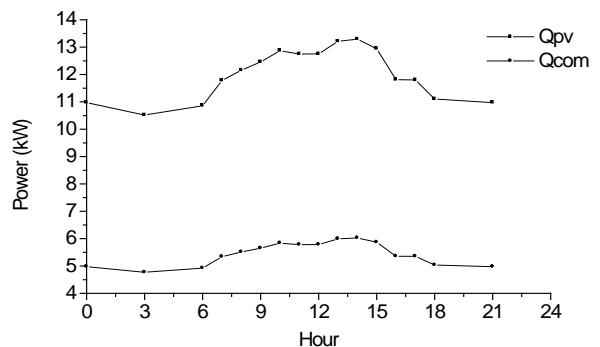


**Figure 14:** Hourly variation of PV energy for the 1<sup>st</sup> days of the months from March to October 2012, for an evaporating temperature  $T_e= 0^\circ\text{C}$



**Figure 15:** Hourly variation of PV energy for evaporating temperatures of 0°C, +10°C, +5°C, -5°C and -10°C on 1<sup>st</sup> April 2012

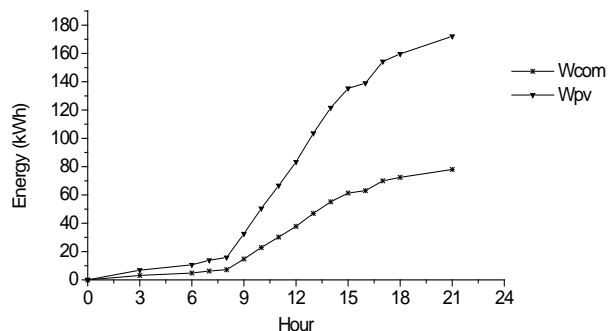
Figure 16 shows the evolution of the output of the PV panels and that absorbed by the compressor on April for an evaporating temperature of 0 °C. It could be seen that the energy provided by the PV panel is greater than that consumed by the compressor; it has a minimum value of 9.93 kW and a maximum value of 12.77 kW, while the power absorbed by the compressor varies from 4.8 to 6.18 kW.



**Figure 16:** Comparison between the hourly power consumption of the compressor and the PV power generated on April 1st, 2012 for an evaporating temperature  $T_e= 0^\circ\text{C}$

Figure 17 presents a comparison between the energy absorbed by the compressor and the energy of PV panels in April for an evaporating temperature of 0 °C. It could be

checked that, between 4 pm and 9 pm, energy exceeds consumption. So the excess energy could be stored in batteries for latter uses at operating hours of low energy such as 8 am, 9 am, 4 pm, 5 pm and 6 pm.



**Figure 17:** Comparison between hourly energy consumption of the compressor and the PV energy generated on April 1st, 2012 for an evaporating temperature  $T_e = 0^\circ\text{C}$

#### 4.6. Evaluation of the overall efficiency of the system

Considering the values of the total heat load and PV power calculated from March to October the overall monthly efficiencies of the system are presented in table 10. It could be seen that the overall performance of the plant also varies with season. In the warmer months, we have higher values. We can therefore conclude that, our SE-VCR system will have better returns for the months of March, April and May, with a maximum value of 35%.

### 5. Conclusions and Recommendations

The overall objective of this work was to contribute to the improvement of thermal comfort in living environments by using a cooling system coupled to a solar installation. An evaluation of the hourly cooling load was done by the means of heat balance due to walls, solar radiation, air renewal, occupants, lighting and electrical appliances. The variations of the various parameters were calculated as well as the minimum area of the photovoltaic panel to satisfy the power demand of the compressor. The results showed that the effective power of the compressor varies between 5.33 kW and 6 kW; the coefficient of refrigerating performance varies from 3.28 to 3.74 while the efficiency of the installation varies from 17 % to 35 %. Finally, the proposed SE-VCR system can successfully be operated in dry tropical areas to improve thermal comfort. The results obtained in this paper could be used in some other areas like Niamey in Niger; Ndjamena in Chad; Ouagadougou in Burkina Faso and other regions with same climatic characteristics. However, further investigations should be made to see the possibility of using adsorption refrigeration systems in the Sudano-Sahelian areas.

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